LECTURENOTES

ON

ELEMENTSOFMECHANICALENGINEERING



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THERMODYNAMICS

Thermodynamics:

Itmaybedefinedasthesciencewhichdealswiththeconversionofheatintomechanical work or energy by using a suitable medium.

ThermodynamicSystem,BoundaryandSurrounding:

System: Asystemisdefined as any quantity of matter or a region in space having certain volume upon which our attention is concerned in analysis of problem.

Surrounding: Anythingexternaltothe systemconstitute as surrounding.

Boundary: System isseparated from the surrounding by system boundary. This boundary may be fixed or movable.

Systemsare classified into three types as follows.

OpenSystem:

It is also known as *flow system*. Open system is one in which both mass and energy crosses the boundary. Open system is also called control volume. Ex- reciprocating air compressor, turbine, pump etc.

Closed System:

It is also known as *non-flow system*. In this system the mass within the boundary remains constant only energy interaction takes place with respect to the surrounding. Ex – Cylinder piston arrangement, Tea kettle.

IsolatedSystem:

An isolated system is one in which there is no interaction between the system and surrounding. There is no mass and energytransfer across the system. Ex- Universe.

ThermodynamicProperty:

Intensive property: The properties which are independent of mass of the system are known as intensive properties. Its value remains the same whether one considers the whole system or only a part of it. The intensive property includes pressure, temperature, specific volume, specific energy, specific density etc.

Extensive property: the property which depends upon mass of the system are known as extensive property. The extensive properties include volume, energy, enthalpy, entropy etc.

Firstlawofthermodynamics:

This lawmaybestatedasfollow:

Theheat and mechanicalworkaremutuallyconvertible. Accordingtothislaw,whenaclosed system undergoes a thermodynamic cycle, the net heat transfer is equal to the net work transfer. In other works, the cyclic integral of heat transfers is equal to the cyclic integral of work transfers.

Mathematically,
$$\oint \delta Q = \oint \delta W$$

Where $\phi \delta Q$ and $\phi \delta W$ represent infinitesimal elements of heat and work transfers respectively.

Theenergycanneitherbecreatednordestroyedthoughit canbetransferredfromone formto another. According to this law, when a system undergoes a change of state, then both heat transfer and worktransfer takesplace. The net energytransfer is storedwithinthe systemand is known as stored energy or total energy of the system.

Mathematically
$$dQ - dW = dU$$

Whenever heat is absorbed by a system it goes to increases its internal energy plus to do some external work (PdV work) i.e. $Q = \Delta U + W$

Where Q is the energy entering a system, ΔU is the increase in internal energy, W is the external work.

$Q = \Delta U + PdV$

Heat:

- ❖ Heat is a transfer form of energy that flows between two systems or between system and surrounding due to temperature difference between them.
- \diamond Amount ofheattransferbetween state 1 and 2 is given by Q_{1-2} .
- ❖ TheS.IunitofheatisJoule(J)orKiloJoule(kJ)andinM.K.SitisgivenbyCalorie.
- Heat transfer to a system is considered as positive and heat transfer from a system is considered as negative.
- Heatisaformofenergyintransit.
- **!** It is a pathfunction.
- Itis lowgrade energy.
- **!** Itisaboundaryphenomenon.
- Itisassociatedinaprocessnotinastate.

Work:

- Work is said to be done when the point of application of a force on a body results in its motion.
- ❖ Workisdefinedastheproductoftheforce(F)andthedistancemoved(s)inthe direction the force. Mathematically, work done, W= Fx s,
- ❖ The mechanicalworkonabody is the amount of mechanical energy transferred to the body by a force.
- \diamond Amount ofworktransferbetweentwo state 1 and 2 is given by W_{1-2} .
- ❖ TheS.I unitofworkisJoule(J)or kilojoule(kJ).
- Work transfer to a system is considered as negative and work transfer from a systemis considered as positive.
- ❖ Workisformenergyin transit.
- ❖ It is a pathfunction.
- ! Itishighgradeenergy.
- Itisaboundaryphenomenon.

Itisassociatedinaprocessnotinastate.

Specificheat:

- * It is the amount of heatenergy required to change the temperature of unitmass of asubstance by one degree.
- * It is classified as specific heat at constant pressure (Cp) and specific heat at constant volume (Cv).
- ♣ Cp–Cvis equaltoR.

Specificheatatconstantvolume:

Itistheamountof heatrequiredonetoraisethetemperatureof aunitmassof gasthrough one degree when it is heated at a constant volume. It is generally denoted by Cv.

Specificheatatconstantpressure:

It is the amount of heat required to raise the temperature of a unitmass of a gas through one degree, when it is heated at constant pressure. It is generally denoted by Cp.

RelationbetweenCpandCv:

Consideragasenclosedinacontainerandbeingheated, at a constant pressure, from the initial state 1 to the final state 2.

m=Massofthe gas

T₁=Initialtemperatureofthegas

 T_2 = Final temperature of the gas

 V_1 = Initial volume of the gas

 V_2 = Final volume of the gas

Cp=Specificheatatconstantpressure

Cv= Specific heat at constant volume

P= Constant pressure

We know that heat supplied to the gas at constant pressure, Q_{1-2} = m.Cp. (T_2-T_1)(i) A part of this heatis utilised in doing the external work, and the rest remains within the gas and is used in increasing the internal energy of the gas.

$$\label{eq:heatutilised} Heatutilised for external work W_{1-2}=P(V_2-V_1). \tag{ii}$$

$$\label{eq:heatutilised} Increase in internal energy dU=mCv(T_2-T_1). \tag{iii}$$

$$\label{eq:heatutilised} We known that Q_{1-2}=W_{1-2}+dU. \tag{iv}$$

$$\label{eq:heatutilised} Or \qquad m.Cp \ (T_2-T_1)=P(V_2-V_1)+m.Cv(T_2-T_1). \tag{v}$$

Using characteristic gas equation (i.e. PV=mRT), we have PV₁=

$$mRT_1$$
 and $PV_2 = mRT_2$

Thus
$$P(V_2-V_1)=m.R(T_2-T_1)$$

Nowsubstituting the value of P(V₂-V₁) in equation (v), we get

$$m.Cp(T_2-T_1)=m.R(T_2-T_1)+m.Cv(T_2-T_1)$$

or
$$C\mathbf{p} = R + C\mathbf{v}$$

The equation (v) gives an important result as it proves that characteristic constant of a gas

(R)isequaltothedifferenceofitstwo specificheatsi.e.(Cp-Cv).

The above equation may be rewritten as

$$C\mathbf{p}$$
– $C\mathbf{v}$ =R,Dividing $C\mathbf{v}$ inbothsidesweget

or
$$Cp/Cv-1=R/Cv$$
 where $Cp/Cv=\gamma$

or
$$\gamma$$
-1= R/Cv

or
$$Cv=R/(\gamma-1)$$

LawofPerfectgases:

1) Boyle's Law:

The volume of a given mass of gas is inversely proportional to its absolutely pressure at constant temperature.

i.e
$$V\alpha^{1}_{p}$$
 (T=constant), PV=Constant.

2) Charles's Law:

The volume of a given mass of a gas is directly proportional to its absolute tempat constant pressure.

i.e
$$V\alpha T$$
 (P=constant), V/T =Constant.

3) Gay-LussacLaw:

The pressure of a given mass of a gas is directly proportional to its absolute temp at constant volume.

i.e
$$P\alpha T$$
 (V=constant), P/T =Constant.

4) IdealgasLaw:

FromBoyle'slaw PV=C

FromCharles'slaw V/T=C

Combiningboth the laws: $\frac{PV}{T} = C$, i.ePV=RT

This equation is called characteristic gas equation or ideal gas equation. Where R= characteristic gas constant = 0.287~KJ~/~Kg-k~(for~atm.~air)

5) Avogadro'slaw:

Itstatesthattheequalvolumesofdifferent idealgasesareatthesametemperature& pressure contains equal number of molecules.

PROPERTIESOFSTEAM

Steam:

Steamis the gaseousphaseofwater.

Saturatedsteam:

* Thesteamwhichisabouttocondenseiscalledassaturated steam.

Wet steam:

- ♣ Itisthemixtureofdrysteamand waterparticlesasmoisture.
- ♣ The enthalpy of wet steam is given by: $h = h_f + x.h_{fg}$ where 'x' is the dryness fraction of steam.

Drysteam:

- ♣ Itisthesaturatedvapor freefrommoisture.
- ♣ The drysaturated steam has absorbed its full latent heat. The enthalpy of drysaturated steam is given by: $h = h_g = h_{f} + h_{fg}$ where dryness fraction x = 1.

Superheatedsteam:

- * Itisthesteamexisting athigher temperaturethanitssaturationtemperature.
- ♣ Thetotalheatrequiredforthesteamtobesuperheatedish_{sup}=totalheatfordry steam+ heat for superheated steam = h_f + h_{fg} + Cp (t_{Sup} -t) = h_g + $Cp(t_{Sup}$ -t)

Where $C\mathbf{p}$ =mean specific heat at constant pressure for superheated steam. t_{sup} = temperature of the superheated steam.

t=saturationtemperatureatthegivenconstant pressure. The difference (t_{Sup} -t) is known as degree of superheat.

Drynessfraction:

♣ Itis themeasure of quality of wetsteam. It is the ratio of mass of dry steam (mg) to the total mass of wet steam (m).

Enthalpy ofsteamorTotal heat:

 \clubsuit It is the sum of enthalpy of saturated water (h_f) and enthalpy of vaporization (h_{fg}).

Sensibleheat(enthalpyofsaturatedwater-h_f):

♣ Sensible heat isdefined asanamount ofheat energyabsorbed by 1 kg ofwater during its heating from 0°C to the saturation temperature at a given pressure.

Latentheatoffusion:

• It is defined as the amount of heat required to convert one kg of ice into water at constant temperature of 0° C.

Steamtableand theiruses:

♣ The properties of dry superheat steam like its temperature of formation (saturation temperature), sensible heat, latent heat ofvaporization, enthalpyor totalheat, specific volume, entropyetc., varywithpressureand can be found byexperimentsonly. These properties have been carefully determined and made available in a tabular formknown as steam tables.

(Solve Numerical to determine Total heat of Wet, Dryand Superheated Steam)

BOILER

Steamgenerator:

- Aboilerisasteamgenerator which is used to convert steamfromwater by heating it.
- A steam generator or boiler, usually, a closed vessel made of steel. Its function is to transfer the heat produced by the combustion of fuel (solid, liquid or gas) to water, and ultimately to generate steam. The steam produced may be supplied:
 - o Toanexternalcombustionengine, i.e. steamengines and turbines.
 - Atlow pressures for industrial process work in cotton mills, sugar factories, breweries etc.
 - o Forproducinghotwater.

Classification of steamboilers:

1. According to the contents in the tube

- (a) Firetubeor smoketube boiler
- (b) Watertubeboiler.

2. According to the position of the furnace

- (a) Internally fired boilers
- (b) Externally fired boilers

3. According to theaxisoftheshell

- (a) Verticalboilers
- (b) Horizontalboilers

4. According to the number of tubes

- (a) Singletubeboilers
- (b) Multitubularboilers

$5. \ According to \ the method circulation of water and \ steam$

(a) Natural circulation boilers

(b) Forcedcirculationboilers

6. According to the use

- (a) Stationaryboilers
- (b) Mobileboilers

Firetube boiler:

- Fire tube boilers are those in which flue gas flows inside the tubes and the water that is to be heated remains outside the tube.
- ♣ Examples of fire tube boilers are: Simple vertical boiler, Cochran boiler, Lancashire boiler, Cornish boiler.

Watertubeboiler:

- Watertube boilers arethose inwhichwaterflowsinside thetube and flue gasremains outside the tube to heat the water.
- ♣ Examples of water tube boilers are: Babcock and Wilcox boiler, Stirling boiler, La-Mont boiler, Benson boiler.

COCHRAN BOILER:

Cochran boiler is a vertical multi-tubular fire tube boiler. It produces steam at low pressure from the heat exchange between water and flue gas. It has the Steam capacity up to 3500 kg/hr.

Construction of Cochranboiler:

It consists of a cylindrical shell with a dome shaped top where the space is provided for steam. The furnace is one piece construction and is seamless. Its crown has a hemispherical shapeandthusprovidesmaximumvolumeofspace. It has the following parts and mountings.

- ❖ Boilershell(cylindrical,topisdomeshaped, hemisphericalcrown)
- Grateandfurnace(Internallyfiredboiler)
- Combustionchamberand fire tubes
- Smokeboxand chimney

* Mountings: water gauge, pressure gauge, fusible plug, feed check valve, steam stopvalve, safety valve and blow off cock.

WorkingofCochran boiler:

When the fuel burns inside the fire box/furnace flue gas produces and flows into the combustion chamber after striking through the fire brick linings. Then the flue gas passes through the fire tubes to exchange heat with water surrounding to them. Then the flue gas is collected in a smoke box and escape to the atmosphere through chimney. In this way the steam produces at the top of the boiler shell and collected.

<u>Mountings</u>: These are the fitting and devices which are necessary for the operation and safety of a boiler.

SteamStopValve:Itisuse toregulate the flow ofsteamfromthe boilertothe steampipe.

❖ Safety Valve: Itisuseforreleasingtheexcesssteamwhenthepressure of

steaminsidetheboilerexceedstheratedpressure. Typesof safety valve are the following: • Dead weight safety valve, Lever safety valve, Springloaded safety valve, Gravity safety valve

❖ WaterLevelIndicator:Itisuseto indicatethelevelofwater intheboilerconstantly.

PressureGauge: It is use to measure the pressure exerted in side the vessel.

Fusible Plug: It is use to protect the boiler against damage due to overheating

for low water level.

❖ FeedCheckValve: Itisuse to controlthesupplythewatertothe boilerandto

prevent the escaping ofwater from the boiler whenthe pump is

stopped.

❖ BlowOffCock: Itisuseto dischargeaportionofwaterwhentheboileris

emptywhennecessaryforcleaning,inspection,repair,mud, scale

and sludge.

❖ ManHole: Itisusedforinspectionandmaintenancepurpose.

<u>Accessories</u>: These are auxiliaryparts required for steamboilers for the proper operation and for the increase of their efficiency.

❖ Super heater: It is use to increase the temperature of steam above it saturation

point.

Economizer: Itisadevice inwhichthewasteheat offluegasesisutilized for

heating the feed water before supplying into the boiler.

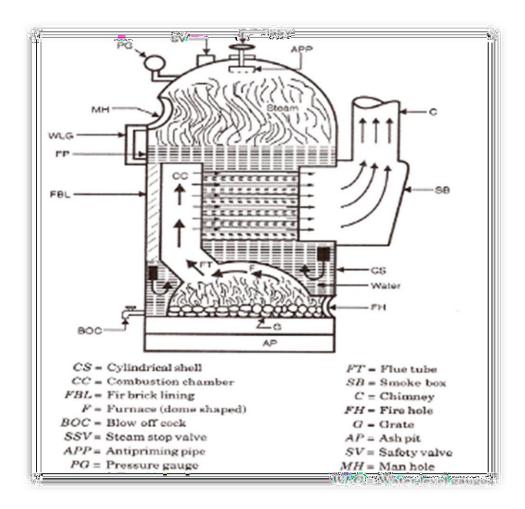
Air pre heater: It is use to increase the temperature of airbeforeitenters the

furnace.

ESP: Itisusedto collect dustorharmfulparticle fromfluegasbefore

escape into the atmosphere.

❖ Boilerfeed pump: Itisused todeliverfeedwatertotheboiler.\



BABCOCKWILCOX BOILER:

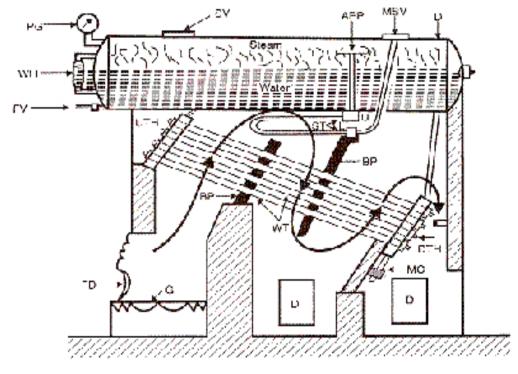
It is a horizontal inclined tube, water tube boiler. In this boiler high pressure steam produces from the heat exchange between water and hot flue gas.

Construction:

It consists of a longitudinal drum connected to a series of front end and rear end header by shortrisertubes. Theseheadersareconnectedbya seriesofinclinedwatertubes. Theangleof inclination of the water tubes to the horizontal is about 15° or more. Mountings are mounted over the boiler shell for safe operation such as: steam stop valve, safety valve, water level indicator, pressure gauge, thermometer, fusible plug, feed check valve, blow- off cock, man hole etc.

Working:

Fuelis supplied to gratethroughfire doorwhere it is burnt. The hot gases are forcedto move upwards betweenthe tubes bybaffle plates provided. The water from the drumflows through the inclined tubes via downtake header and goes back into the shell in the form of water and steamvia uptake header. The steamgets collected in the steamspace of the drum. The steam then enters through the anti-priming pipe and flows in the super heater tubes where it is further heated and is finally taken out through the main stop valve and supplied to the Steam turbine or Steam engine when needed.



B = Brein

DTM - Down take header

WT = Water tubes

BP = Baffle plates

 $D = \mathbf{Dcore}$

G = Grate

 $FD = Fixe \operatorname{door}$

WU = Mort rollectorWLI = Water level indicator PG = Prossure gauge

S7 * Superheater to bes

SV = Safety valve

MSV - Main stop valve

APF = Aptiphrotogrape

L = Lower junction box U = Upper junction box

FV = Feed value

DifferencebetweenWatertubeandFiretubeboiler:

Watertubeboiler

- The water circulates inside the tubes 1.
 which are surrounded by hot gases from
 the furnace.
- 2. Therateofgenerationofsteamishigh.
- 3. Itgeneratessteamatahigherpressure up to 165 bar.

Firetube boiler

- . The hot gases from the furnace the furnace pass throughthe tubes which are surrounded by water.
- 2. Therateofgeneration of steam is low.
- **3.** Itgeneratessteamatupto24.5bar.

- **4.** Foragivenpower,thefloorarearequired for **4.** the generation is less.
- 4. Foragivenpower,the floorarea required for the generation is more.

5. Theoperating costishigh.

- **5.** Theoperating costisless.
- **6.** Theburstingchance ismore.
- **6.** Theburstingchance isless.
- 7. Itisusedfor largepower plants.
- **7.** Itisnotsuitableforlargepowerplants.

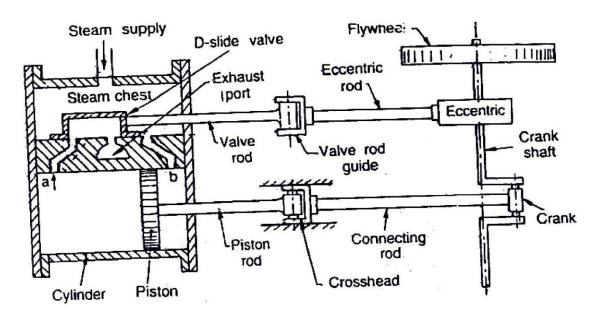
STEAMENGINE

SteamEngine:

In all steam engines, the steam is used as the working substance. These engines operate on the principle offirst law ofthermodynamics, i.e. heat and workare mutually convertible. In a reciprocating steamengine, as the heat energy in the steam is converted into mechanical work by the reciprocating (to and fro) motion of the piston, it is also called reciprocating steam engine.

WorkingofaSingleCylinderDouble Acting HorizontalReciprocating Steam Engine:

The principal parts of a single cylinder, double acting horizontal reciprocating steam engine are shown in Fig.



(Singlecylinder, doubleactinghorizontalreciprocatingsteamengine)

The superheated steam at a high pressure (about 20 atmospheres) from the boiler isled into the steam chest. After that the steam makes its way into the cylinder through any of the ports 'a' or 'b' depending upon the position of the D-slide valve. When port 'a' is open, the steamrushes to the left side ofthe piston and forces it to the right. At this stage, the slide valve coversthe exhaust port andthe other steamport 'b'asshown inFig. Since the pressure of steam is greater on the left side than that on right side, the piston moves to the right.

When the piston reaches near the end of the cylinder, it closes the steam port 'a' and exhausts port. The steam port 'b' is now open, and the steam rushes to the right side of the piston. This forces the piston to the left and at the same time the exhaust steam goes out through the exhaust pipe, and thus completes the cycle of operation. The same process is repeated in other cycles of operation, and as such the engine works.

ImportantTermsusedinSteamEngines:

- 1. **Bore:-** Theinternaldiameterofthecylinderoftheengineisknownasbore.
- 2. **Dead centre's:-** The extreme positions of the piston inside the cylinder during its motion are known as dead centre's. There are two dead centres, i.e. Inner dead centre (I.D.C.) and Outer dead centre (O.D.C.).
- 3. **Clearance volume** The volume of space between the cylinder cover and the piston, when the piston is at I.D.C. position is called clearance volume It is usually represented as a percentage of stroke volume.
- **4. Strokevolumeorsweptvolume-**Thevolumesweptbythepistonwhenitmovesfrom I.D.C. to O.D.C., is known as stroke volume or swept volume v**5**. It is also known as piston displacement. Mathematically, stroke volume or swept volume

$$V_S = \frac{\pi}{4} D^2 L$$

 $\label{eq:whereD=Boreorinternal diameter of the cylinder, and $L = $$ Length of the stroke.$

5. Cut-off volume:- Theoretically, the steam from the boiler enters the clearance space and pushes the piston outward doing external work. At some pointduring outward movement of

the piston, the supplyofsteamis stopped. The point orthe volume where the cut-offofsteam takes place is called the point of cut-off or cut-off volume.

6. **Averagepistonspeed-**Thedistancetravelledbythepistonperunittimeisknownas average piston speed. Mathematically,

 $\label{eq:local_equation} Average piston speed = LNm/min, for single acting steamengine $$=2LNm/min, for double acting steamengine $$$ where L = Length of the stroke in metres, and N = Speed in R.P.M.

7. Mean effective pressure- The average pressure onthe pistonduring the working stroke is called mean effective pressure. It is given by the mean depth of the p-v diagram.

 $Mathematically, mean effective pressure P_m = Work done per cycle/Stroke volume. \\$

Indicator Diagram of a Simple Steam Engine:

It is a graphical representation of the variation in pressure and volume of steam inside the cylinder or p-vdiagram. As a matter of fact, the theoreticalor hypothetical indicator diagram of a simple steam engine has been developed from that of a modified Rankine cycle.

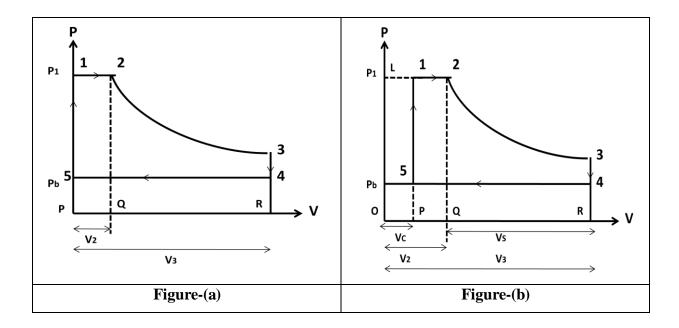
Theoreticalor Hypothetical Indicator Diagram:

The theoretical or hypothetical indicator diagram without clearance and withclearance is shown in Fig. 4.3. Inother words, ifthere is no steam in the cylinder (or there is zero volume of steam at point 1).

Thesequenceofprocessesisgiven below:

1. **Process 1-2 -** At point 1, the steam is admitted into the cylinder throughthe inlet port. As the piston movestowards right, therefore the steam is admitted at constant pressure. Since the supply of steam is cut off at point 2, therefore this point is known as cut-off point.

- 2. **Process 2-3-** At point 2, expansion of steam, in the cylinder, starts with movement of the piston till it reaches the dead end. This expansion takes place hyperbolically (i.e. pv=C) and pressure falls considerably as-shown in Fig. 4.4.
- 3. **Process 3-4** At point 3, the exhaust port opens and steam is released from the cylinder to the exhaust. As a result of steam exhaust, pressure in the cylinder falls suddenly (without change in volume) as shown in Fig. 4.4. The point 3 is known as release point.
- 4. **Process 4-5** At point 4, return journey of the piston starts. Now the used steam is exhausted at constant pressure, till the exhaust port is closed, and the inlet port is open. The steam pressure at point 4 is called back pressure.
- 5. **Process 5-1-** At point 5, the inlet port is opened and some steam suddenly enters into the cylinder, which increases the pressure of steam (without change in volume). This process continues till the original position is restored.



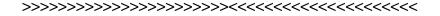
Theoreticalor Hypothetical Mean Effective Pressure:

Theoretical or hypothetical mean effective pressure is given by:

$$\begin{split} \boldsymbol{P}_{m} &= \frac{\textit{Workdonepercycle}}{\textit{StrokeVolume}} = \frac{P_{1}V_{2} + 2.3P_{1}V_{2}log(\frac{V_{3}}{V_{2}}P_{b}V_{3}}}{V_{3}} = P \\ & \frac{V_{2}}{1_{V_{3}}} + 2.3P \\ & \frac{V_{2}}{1_{V_{3}}}log^{V_{2}} - P \\ & \frac{P_{1}}{V_{3}}(1 + 2.3logr) - P \\ & r \end{split}$$
 Where, $r = \frac{V_{3}}{V_{2}} = Expansion ratio$

Numerical

- 1) A steam engine cylinder receives steam at a pressure of 11.5 bar and cut-off takes place at half of the stroke. Find the theoretical mean effective pressure, if the back pressure of the steam is 0.15 bar. Neglect clearance.
- 2) The steam is supplied at a pressure of 8.4 bar and cut-off occurs at 0.35 ofthe stroke. The back pressure is 1.25bar. If the diagramfactor is 0.75, determine the actualmean effective pressure. Neglect clearance.
- 3) Adoubleactingsinglecylinderhas200mmstroke,160mmdiameter.Itrunsat250 r.p.mandthecut-offis25% ofthestroke.Thepressureatcut-offis15barand exhaust isat 0.3 bar for adiagram factorof0.75.Estimate the indicated power inkW.
- Calculate the indicated power and steam consumption in kg/h of a double actingsteam engine from the following data:
 Diameter of cylinder=300 mm; stroke=450mm; R.P.M=120; Steam pressure=7 bar, and 0.9 dry; Back pressure = 1.2bar; Cut-off takes place at 32% of stroke for both ends.
- 5) Duringatestonasingleactingnon-condensing, singlecylindersteamengine, the following observations were recorded:
 - Bore-225mm;Stroke-600mm;Speed=100r.p.m.;Effectivebrakediameter= 2.75m;Netload on thebrake=1650N; Areaofindicatordiagram =2500mm²; Length of indicator diagram = 100 mm; Spring strength = 530 bar/m.
 - Determine: 1. Indicated power; 2. Brakepower; and 3. Mechanical efficiency.
- 6) Estimatethebrakepowerofsimplesteamengine having 250 mmpiston diameter, and 40 mm piston rod diameter with 250mm stroke length operating at 300 r.p.m. The initial and back pressure of steam is 8.5 bar and 1.2 bar respectively. Assume 90% mechanical efficiency, cut-off at 25% of the forward stroke and 0.73 diagram factor. Neglect clearance and compression.



Powerdeveloped byaSimpleSteam Engine:

Powermaybedefinedastherateofdoingwork. It is thus the measure of performance of a steam engine.

Mathematically: Power(P)=(Workdone) / (timetaken)

Incase of steamengines, the following two terms are commonly used for the power developed.

1. Indicatedpower, and 2. Brakepower

Indicated power:

Actualpowergenerated intheenginecylinderiscalledpowerinputorindicated power.

For single acting steamengine: $I.P = {}^{P_m LA\underline{N}}$ Watt

For doubleacting steamengine: $I.P = \frac{2P_mLA_N}{60}$ Watt

STEAMTURBINE

TURBINE:

Asteamturbine is a machinewhich converts the available thermal energy into mechanical energy. These are classified as Impulse and Reaction turbine.

Typesofturbine(Classification):

Thesteamturbines may be classified into the following types:

- 1. According to themodeofsteamaction
 - (i) Impulseturbine, and
- (ii)Reactionturbine
- 2. According to the direction of steamflow
 - (i) Axialflowturbine, and
- (ii)Radialflowturbine.
- 3. According to the exhaust condition of steam
 - (i) Condensingturbine, and
- (ii)Non-considering turbine
- 4. According to the pressure of steam
 - (i) Highpressure turbine,
- (ii)Mediumpressureturbine, and
- (iii)Lowpressureturbine
- 5. According to the number of stages
 - (i) Singlestageturbine, and
- (ii) Multistageturbine

${\bf Difference between Impulse and reaction turbine:}$

Impulseturbine

Reactionturbine

- Thesteamflowsthroughthenozzlesand impinges on the moving blades.
- The steam flows first through guide mechanismandthenthroughthemoving blades.
- Thesteamimpingesonthebucketswith kinetic energy.
- 2. Thesteamglidesoverthemovingvanes with pressure and kinetic energy.
- **3.** Thesteammayormaynotbeadmitted over the whole circumference.
- **3.** Thesteammustbeadmittedoverthe whole circumference.

- 4. Thesteampressureremainsconstant during 4.its flow through the moving blades.
- **4.** Thesteampressureisreducedduringits flow through the moving blades.
- 5. The negative velocity of steam while glidingoverthebladesremains constant.
- 5. The relative velocity of steam while glidingoverthemovingbladesincrease
- **6.** Thebladesaresymmetrical.
- **6.** Thebladesarenot symmetrical
- **7.** Thenumberofstagesrequiredislessfor the same power developed.
- **7.** Thenumberofstagesrequiredismore for the same power developed.

STEAMCONDENSER

CONDENSER:

A steamcondenser is a closed vessel into which the steam is exhausted, and condensed after doing work in an engine cylinder or turbine. A steam condenser has the following twoobjects.

- 1. The primary object is to maintain a low pressure (below atmospheric pressure) so as to obtain the maximum possible energy from steam and thus to secure a high efficiency.
- 2. The second object is to supply pure feed water to the hot well, from where it is pumped back to the boiler.

Classification of Condensers:

The steam condensers may be broadly classified into the following two types: (1) Jet condensers or mixing type condensers; (2) Surface condensers or non-mixing typecondensers.

TypesofJetCondensers:

- 1. Parallelflowjetcondenser
- 2. Counterfloworlow leveljetcondenser
- 3. Barometricorhighleveljetcondenser, and
- 4. Ejectorcondenser.

TypesofSurface Condensers:

- 1. Downflowsurfacecondenser
- 2. Centralflowsurfacecondenser
- 3. Regenerativesurfacecondenserand
- 4. Evaporative condenser.

INTERNAL COMBUSTIONENGINE

InternalCombustion(I.C) engine:

- ♣ Whenthecombustionoffuelsuppliedtotheenginetakesplaceinsidetheengine cylinder, the engine is called as an internal combustion engine.
- ♣ Examples— 2-strokeand 4-strokepetroland diesel engines.

Terminology:

- **Bore:** It isthediameteroftheenginecylinderorpiston.
- * *Stroke*: Itisthedistancemovedbypistonbetweentwodeadcentres.
- * <u>Sweptvolume</u>: It is the maximum volumes wept by the piston when it moves from one dead centre to another.
- Compression ratio: It is the minimum volume between the cylinder head and top of the piston when the piston is at the top dead centre.
- Airstandardefficiency: Itisratiobetweenworkdoneandheatsuppliedforair standard cycle.

MainComponents of I. Cengine:

Themaincomponents of an I. Cenginear ecylinder, cylinder head, piston, piston rings, connecting rod, crank, crank shaft and flywheel.

ClassificationofI.C engine:

The classification of I. Cengine is as follows:

- ♣ Accordingtonumberofstrokes:
 - o Fourstrokeengine
 - o Twostrokeengine
- Accordingtofuelused
 - Petrolengine
 - o Dieselengine
 - o Gasengine
- According to method of ignition
 - Sparkignitionengine

- o Compressionignitionengine
- Accordingtocoolingsystem
 - o Aircooledengine
 - o Watercooledengine
- Accordingtonumberofcylinder
 - o Singlecylinderengine
 - o Multicylinderengine
- Accordingtospeedofengine
 - o Lowspeed engine
 - o Mediumspeedengine
 - o Highspeedengine

Difference between Petrolengine and Dieselengine:

Petrolengine

- **1.** Airfuelmixtureisdrawnintothe enginecylinderinsuctionstroke.
- 2. The carburetorisus ed to mixair and petrol in a proper ratio.
- **3.** Pressureattheendofcompressionis about 10 bar.
- **4.** Theairfuelmixtureisignitedby using the spark plug.
- Combustionoffueltakesplaceat constant volume.
- **6.** Ithascompression ratio approximately 6 to 10.

Dieselengine

- **1.** OnlyAirmixtureisdrawnintothe engine cylinder in suction stroke.
- 2. Fuelinjectorisusedtoinjectfuel.
- **3.** Pressureattheendofcompressionis about 35 bar.
- 4. Mixtureisselfignitedathigh compression ratio.
- **5.** Combustionoffueltakesplaceat constant pressure.
- 6. Ithascompression ratio approximately 15 to 25.

- 7. It haseasystarting.
- **8.** Itislighterandcheaper.
- 9. Itsrunningcostishigh.
- 10. Itsmaintenancecostis less.
- 11. Ithasthermalefficiencyup to 26%.
- 12. Itisusedinlightvehicles.

- 7. Itsstartingisdifficult.
- **8.** Itisheavierand costlier.
- 9. Itsrunningcostislow.
- 10. Itsmaintenancecostishigh.
- 11. Ithasthermalefficiencyup to 40%.
- 12. Itisusedinheavyvehicles.

Differencebetween2-strokeand4-strokeengine:

2-Strokeengine

- Twostrokeenginesgiveonepower stroke for eachrevolutionofcrank.
- Power produced by this engine is almostdoublethanfourstrokecycle engine.
- **3.** Toproducesamepoweritrequires less space.
- 4. Ithasinlet, exhaust and transfer ports.
- 5. Itsthermalefficiencyislow.
- **6.** Itrequireslighterflywheels.
- 7. Itisusedinlightvehicles.
- **8.** Itsinitialcostisless.

4-Strokeengine

- **1.** Four stroke engines give one power strokeforeverytworevolutionsofcrank.
- 2. Powerproducedbythisengineisalmost half than two stroke cycle engine.
- **3.** Toproducesamepoweritrequiresmore space.
- 4. Ithasinletandoutletvalves.
- 5. Itsthermalefficiencyishigh.
- **6.** Itrequireslarger flywheels.
- 7. Itisusedinlight,mediumandheavy vehicles.
- **8.** Itsinitialcostishigh.

IndicatedhorsePowerofI.Cengine:

Indicated power is defined as the rateofwork done on the piston by the combustion of charge inside the engine cylinder. Indicated power in terms of horse power is called as *Indicated horse power*.

Mathematically:
$$I.P = \frac{P_m LAnk}{60}$$

Brakehorse powerofI.C engine:

Itisthenet poweravailableattheengineshaft.Brakepowerintermsofhorsepoweriscalled as *Brake horse power*.

Mathematically:
$$B.P = \frac{2\pi RNF}{60} = \frac{2\pi NT}{60}$$

Mechanical efficiency of I. Cengine:

Itisdefinedastheratioofbrake powertothe indicatedpowerorI.H.PtoB.H.P.

FourstrokeCyclePetrolEngine:

It is also known as Otto cycle. It requires four strokes of the piston to complete one cycle of operation in the engine cylinder. The four stroke of a petrol engine sucking fuel air mixture (petrol and Air operation with proportionate quantity of air in the carburetor known ascharge) are described below:

1. Suctionorchargingstroke

In this stroke, the inlet valve opens and charge is sucked into the Cylinder as the pistonmoves downward fromtop dead centre(T.D.C). It continues tillthe piston reaches Its bottom dead centre (B.D.C).

2. Compressionstroke

In this stroke, both the inlet and exhaust valves are closed and charge is compressed as the piston moves upwards from B.D.C. to T.D.C. As a result of compression the pressure and temperature of the charge increases considerably (the actual values depend upon compression ratio). This completes one revolution of the crankshaft.

3. Expansionorworkingstroke

Shortly before the piston reaches T.D.C. (during compression stroke), the charge is ignited with the help of a spark plug. It suddenly increases the pressure temperature of the productsof combustion but the volume, practically, remains constant. Due to rise in pressure, the piston is pushed down with a great force. The hot burnt gases expand due high speed of the piston. During this expansion, some of the heat energy produced is transformed mechanical work. It may be noted that during this working stroke both valves are closed and pistonmoves from T.D.C. to B.D.C.

4. Exhauststroke

In this stroke, the exhaust valve is open as piston moves from B.D.C to T.D.C. This movement ofthe piston pushes out the productsof combustion, from the engine cylinder and are exhausted through the exhaust valve into the atmosphere.

Nowthecyclecompletesand theenginecylinderisreadytosuckthechargeagain

FourstrokeCycleDieselEngine

It is also known as compression ignition engine because the ignition takes place due to the compression produced in the engine cylinder at the end of compression stroke. The four strokes of a diesel engine sucking pure air are described below:

1. Suctionorchargingstroke

In this stroke, the inlet valve opens and pure air is sucked into the cylinder as the piston moves downwards from the top dead centre {TDC). It continues till the piston reaches its bottom dead centre {BDC).

2. Compressionstroke

In this stroke, both the valves are closed and the air is compressed as the piston moves upwards from BDC to TDC. As a result of compression, pressure and temperature of the air increases considerably(the actualvalue depends uponthe compression to). This completes one revolution of the crank shaft.

3. Expansionorworkingstroke

Shortlybeforethepistonreaches the TDC (during the compression stroke), fueloilis injected in the form of very fine spray into the engine cylinder, through the nozzle, known as fuel injection valve. At this moment, temperature of the compressed air sufficiently high to ignite the fuel. It suddenly increases the pressure and temperature of the producer of combustion. The fuel oil is continuously injected for a fraction of the revolution. The fuel oil assumed to be burnt at constant pressure. Due to increased pressure, the piston is pushed down with a great force. The hot burnt gases expand due to high speed of the piston. During this expansion, someofthe heat energy is transformed into mechanical work. It may be noted that during this working stroke, both the valves are closed and the piston moves from TDC to BDC.

5.Exhauststroke

In this stroke, the exhaust valve is open as the piston moves from BDC to TDC. This movement of the piston pushes out the products of combustion from the engine cylinder through the exhaust valve into the atmosphere. This completes the cycle and the engine cylinder ready to suck the fresh air again.

HYDROSTATICS

Fluid:

Fluid isasubstance havingparticleswhichreadilychangetheirrelative motionandtheyhave the capability of flowing. Example: air and liquid. The standard liquid is taken as water and standard gas is taken as air.

PropertiesofFluid:

1. **Densityormassdensity:**Itistheratioofmassperunitvolumeofafluid.Itis denoted the symbolp (rho).

<u>Unit</u>:Theunitofmassdensityiskg/m³inS.I andgm/cm³inC.G.S

mathematically: $\rho = \frac{massoffluid}{volume\ offluid}$

The value of density of water is 1 gm/cm³ or 1000 kg/m³

2. Specificweightorweightdensity:

Specific weight or weight densityofa fluid is the ratio betweenthe weights of afluid to its volume. Thus weight perunit volume of a fluid is called weight density and it is denoted by the symbol w.

Mathematically: $w=\frac{weightoffluid}{volumeoffluid} = \frac{m \times g}{V} = \rho \times g$

<u>Unit</u>:TheunitofweightdensityisN/m³inS.Ianddyne/cm³in C.G.S

We know that: $1 \text{ liter} = \frac{1}{0.00} \text{ m}^3, 1 \text{ liter} = 1000 \text{ cm}^3$

 $The value of specific weight density (w) for water is 9810 N/m^3 in S.I.\ units.$

3. Specificgravityorrelativedensity(S):

Specificgravity is defined as the ratio of the mass density or weight density of a standard liquid (water at 4^{0} c).

 $Mathematically: S(forliquids) = \frac{weightdensity(density)ofliquid}{weightdensity(density)ofwater}$

Itisunitless.Specificgravityofwater=1 and specificgravityofmercury=13.6

4. Viscosity(μ):

Viscosityisdefined asthepropertyofa fluidwhichoffersresistancetothe movement of one layer of fluid over another adjacent layer of the fluid.

Mathematically:
$$\mu = \frac{\text{shearstress}}{\text{velocitygradient}}$$

<u>Unitsofviscosity</u>:-N-s/m²inS.I,kgf-sec/m²inM.K.S,dyne-sec/cm²inC.G.S itisalsoknownasdynamicviscosity. 1poise=dyne- sec/cm²

Kinematicviscosity:

Itisdefinedastheratio betweenthedynamicviscosityanddensityoffluid. It is denoted by ν (nu).

$$v = \frac{viscosity}{density} = \frac{\mu}{\rho}$$

<u>Unitofkinematic viscosity</u>: m²/secinS.Iandcm²/secinC.G.S

1stoke =cm²/sec=
$$(1 \frac{1}{100})^2$$
m²/sec

Typesof Fluid:

Idealfluid&Realfluid.

Idealfluidoftwotypes-(I)Newtonianfluid(ii) Non- Newtonianfluid.

IdealFluid orPerfectFluid:

Idealfluidsareonlyimaginaryfluidswhichareincompressible,non-viscous,nosurface tension.

Innatureidealfluid doesnotexist.

Realfluid:

The fluid's actually available in nature causing property such as viscosity, surface tension and compressibility.

Pressure:

Pressure is forceper unit areaapplied inadirectionperpendicular tothesurfaceofanobject. The

SI unit of pressure is the newton per meter square, which is called the Pascal (Pa).

Absolutepressureiszero-referencedagainstaperfectvacuum,soitisequaltogauge pressure plus atmospheric pressure.

Gaugepressureiszero-referencedagainstambientairpressure, soitisequal to absolute pressure minus atmospheric pressure.

Vacuumpressureiszero-referencedagainstambientairpressure, soitisequalto atmospheric pressure minus absolute Pressure.

Differentialpressureisthedifferenceinpressurebetweentwopoints.

PressureMeasuringInstruments:

 $In struments used to measure pressure are called {\bf pressure gauges} or {\bf \ vacuum gauges}.$

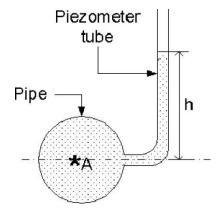
A'*manometer'* isaninstrumentthat usesacolumnofliquidtomeasurepressure. A **vacuum gauge** is used to measure the pressure in a vacuum.

Typesof manometer:

- 1.Simplemanometer
- 2.U-tubemanometer
- 3.Differentialmanometer

Piezometer:

Piezometer is one of the simplest forms of manometers. It can be used for measuring moderate pressures of liquids. The setup of piezometer consists of a glasstube, inserted in the wall of a vessel or of a pipe. The tube extends vertically upward to such a height that liquid can freely rise in it without overflowing. The pressure at any point in the liquid is indicated by the height of the liquid in the tube above that point.



Pressure at point A can be computed by measuring the height to which the liquid rises in the glass tube. The pressure at point A is given by $\mathbf{p} = \mathbf{w.h}$, where w is the specific weight of the liquid.

"U"-TubeManometer:

Using a "U"-Tube enables the pressure of both liquids and gases to be measured with the same instrument. The "U" is connected as in the figure below and filled with a fluid called the *monomeric fluid*. The fluid whose pressure is being measured should have a mass density less than that of the manometric fluid and the two fluids should not be able to mix readily – that is, they must be immiscible.

Pressureinacontinuous staticfluidis the same at any horizontal levelso:

pressure at B = pressure at C
$$p_{\mathcal{B}} = p_{\mathcal{C}}$$

Forthelefthandarm:

pressure at B = pressure at A + pressure due to height h $_1$ of fluid being meas ured $p_B = p_A + \rho g h_1$

Fortherighthand arm

pressure at C = pressure at D + pressure due to height h $_2$ of manome tric fluid $p_C = p_{\rm Amospheric} + \rho_{\rm man}\,gh_2$

Gauge Pressure: $p_A = \rho_{man}gh_2 - \rho gh_1$

HYDROKINETICS

TotalHead of a LiquidParticleinMotion:

The total head of a liquid particle, in motion, is the sum of potential head, kinetic head and pressure head.

Mathematically: $TotalHead, H=Z+V^2/2g+P/W$ mofliquid.

Continuity Equation:

The equation based on the principle of conservation of mass is called continuity equation. Thus for a fluid flowing through the pipe at all the cross-section, the quantity of fluid per section is constant.

Consider two cross-sectionofapipeasshownin figure.

Let V_1 =Average velocity at cross-section 1-1 ρ_1 =

Density at section 1-1

A₁=Areaofpipeatsection1-1

And V_2, ρ_2, A_2 are corresponding values at section, 2-2

Then, rate of flow at section $1-1 = \rho_1 A_1 V_1$

Rateofflowatsection $2-2=\rho_2 A_2 V_2$

According to law of conservation of mass,

Rateofflowat section1-1=Rateofflowat section2-2 p₁ A₁

$$V_1 = \rho_2 A_2 V_2$$

This equation is applicable to the compressible as well as incompressible fluids and is called continuity equation. If the fluid is incompressible, then $\rho_1 = \rho_2$ and continuity equation $A_1V_1 = A_2V_2$

Bernoulli's Equation:

It states that, "for a perfect incompressible liquid, flowing in a continuous stream, the total energy of a particle moves from one point to another."

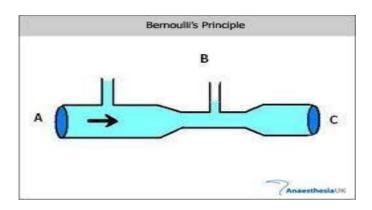
This statement is based on the assumption that there are no losses due to friction in the pipe.

Mathematically,
$$Z + V^2/2g + p/w = constant$$
.

Where, Z=potential energy, V²/2g=Kinetic energy, P/W=pressure energy

Proof:

Consideraperfect incompressible liquid, flowingthroughanon-uniformpipeasshowninthe figure.



Letusconsidertwosections AA&BBofthepipe. Now letus assume that the pipe is running full and there is a continuity of flow between the two sections.

Let Z_1 =HeightofAAabovethe datum.

 P_1 =PressureatAA.

V₁=VelocityofliquidatAA,

 a_1 = cross-sectional area of the pipe at AA, and

 Z_2 , P_2 , V_2 , a_2 = corresponding values at BB.

Let the liquid between the two sections AA and BB moves to A^1A^1 and B^1B^1 through very smalllengthsd l_1 andd l_2 asshowninfig. This movement of the liquid between AA&BB is

equivalenttothemovementoftheliquidbetween AA and A^1A^1 to BB and B^1B^1 the remaining liquid between A^1A^1 and BB being unaffected.

Let Wbetheweight oftheliquidbetweenAAand A¹A¹.Sincethe flow iscontinuous. Therefore W

$$= w.\mathsf{a} 1.\mathsf{d} \mathit{l}_1 = w.\mathsf{a}_2.\mathsf{d} \mathit{l}_2$$

$$\Rightarrow$$
 a₁.d l_1 =a₂.d l_2 =W/w -----(1)

ora₁.d
$$l_1$$
=a₂.d l_2 -----(ii).

We know that work done by pressure at A Ainmoving the liquid to A^1A^1 = force \times distance

$$=p_1a_1.dl_1Si$$

milarly, work done bypressure at BB, in moving the liquid to $B^1B^1 = -P_2.a_2dl_2$ (minus sign is taken as the direction of p_2 is opposite to that of p_1)

Totalworkdonebythepressure= $P_{1.}a_1dl_1-P_{2.}a_2.dl_2$

$$=P_{1.}a_1dl_1-P_{2.}a_1dl_1(a_{1.}dl_1=a_2.dl_2)$$

$$= a_1 dl_1(P_1 - P_2) = W/w (P_1 - P_2)$$
 (: $a_1 dl_1 = W/w$)

Loss of potential energy = w (z_1 - z_2) & gain in kinetic energy = w ($V_2^2/2g$ - $V_1^2/2g$) = $w/2g(Y^2-V^2)$

Weknowthat:lossofpotentialenergy+work donebypressure =gain inkineticenergy W (Z₁-

$$Z_2$$
) + W/w (P₁-P₂) = W/2g (V₂²- V²₁)

$$(Z_1-Z_2)+ P_1/w-P_2/w= V_2^2/2g-V^2/2g_1$$

$$Or \qquad Z_1 + {V_1}^2/2g \ + P_1/w = Z_2 + {V_2}^2/2g + P_2/w$$

ThisprovestheBernoulli's equation.

Assumptions:

The followingaretheassumptionsmade inthederivation of Bernoulli's equation. (i) The fluid is ideal, i.e. viscosity is zero.

(ii) Theflowissteady

(iii) The flow isincompressible (iv) The flowisrotational.

LimitationsofBernoulli'sequation:

The Bernoulli's theorem or Bernoulli's equation has been derived on certain assumptions,

which are rarely possible. Thus the Bernoulli's theorem has the following limitations.

(1) The Bernoulli's equation has been derived under the assumption that the velocity of every

liquidparticleacrossanycross-sectionofapipe is uniform. But inactualpractice, Itisnot so. The

velocity of liquid particle in the centre of a pipe is maximum & gradually decreases towards

the walls of the pipe due to the pipe friction. Thus, while using the Bernoulli's equation, only

the mean velocity of the liquid should be taken into account.

(2) The Bernoulli's equation has been derived under the assumption that no external force,

except the gravity force is acting on the liquid. But in actual practice, it is not so. There is

always someexternalforcesuchaspipe frictionetcactingthe liquid, whichaffectthe flowof the

liquid.

Thus, while using the Bernoulli's equation, all such external force should be neglected. But if

some energy is supplied to, or, extracted from the flow the same should also be taken into

account.

(3) The Bernoulli's equation has been derived, under the assumption that there is no loss of

energyofthe liquidparticlewhile flowing. But inactualpractice, it israrelyso. Inaturbulent flow,

some kinetic energy is converted into heat energy. And in a viscous flow, some energy is lost

due to shear forces. Thus, while using Bernoulli's equation, all such losses should be

neglected.

(4) If the liquid is flowing in a curved path, the energydue to centrifugal force should also be

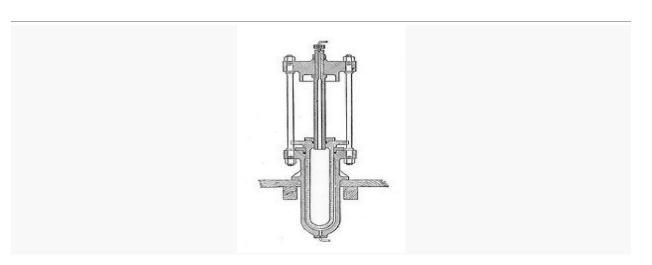
taken into account.

Practical Applications of Bernoull's Equation:

Therearethreetypes: (1)Venturimeter(2)Orificemeter(3)Pitottube.

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Hydraulicintensifier



Such a machine may be constructed by mechanically connecting two pistons, eachworking in a

separate cylinder of a different diameter. As the pistons are mechanically linked, theirforce and stroke length are the same. If the diameters are different, the hydraulic pressurein each cylinder will varyin the same ratio as their areas: the smaller piston giving rise to a higher pressure. As the pressure is inversely proportional to the area, it will be inversely proportional to the *square* of the diameter.

The working volume of the intensifier is limited by the stroke of the piston. This in turn limits the amount of work that may be done by one stroke of the intensifier. These are not reciprocatingmachines(i.e.continually runningmulti-strokemachines)andsotheir entire work must be carried out by a single stroke. This limits their usefulness somewhat,to machines that can accomplish their task within a single stroke. They are often used where a powerful hydraulic jack is required, but there is insufficient space to fit the cylinder size that would normally be required, for the lifting force necessary and with the available system pressure. Using an intensifier, mounted outside the jack, allows a higher pressure to be obtained and thusa smaller cylinder used forthe same lift force. Intensifiers are also used as part of machines such as hydraulic presses, where a higher pressure is required and a suitable supply is already available.

Some small intensifiers have been constructed with a stepped piston. This is a double-ended piston, of two different diameters, each end working in a different cylinder. This construction is simple and compact, requiring an overall length little more than twice the stroke. It is also still necessary to provide two seals, one for each piston, and to vent the area between them. A leak of pressure into the volume between the pistons would transform the machine into an effective single piston with equal area on each side, thus defeating the intensifier effect. A mechanically compact and popular formof intensifier is the concentric cylinder form, as illustrated. In this design, one piston and cylinder are reversed: instead of the large diameter piston driving a smaller piston, it instead drives a smaller moving cylinder that fits over a fixed piston. This design is compact, and again may be made in little over twice the stroke. It has the great advantage though that there is no "pistonrod" and the effective distance between the two pistons is short, thus permitting a much lighter construction without risk of bending or jamming.

Hydraulicaccumulator

A hydraulic accumulator is a pressure storage reservoir inwhicha non-compressible hydraulicfluidisheldunderpressurebyanexternalsource. The externalsource can be aspring, a raised weight, or a compressed gas. An accumulator enables a hydraulic system to cope with extremes of demandusing aless powerful pump, to

respondmorequickly toatemporary demand, and to smooth outpulsations. It is a type of energy storage device.

Compressed gas accumulators, also called hydro-pneumatic accumulators, are by far the most common type.

Hydraulicaccumulator

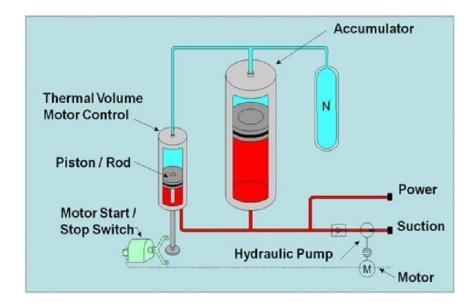
Functioningofan accumulator

In modern, often mobile, hydraulic systems the preferred item is a gas charged accumulator, but simple systems may be spring-loaded. There may be more than one accumulatorina system. The exact type and placement of each may be a compromise due to its effects and the costs of manufacture.

An accumulator is placed close to the pump with a non-return valve preventing flow back to the pump. In the case of piston-type pumps this accumulator is placed in the ideallocation absorb pulsations of energy from the multi-piston pump. It also helps protect the system from fluid hammer. This protects system components, particularly pipe work, from both potentially destructive forces.

An additional benefit is the additional energy that can be stored while the pump is subject to low demand. The designer can use a smaller-capacity pump. The large excursionsofsystem components, such as landinggear on a large aircraft, that require a considerable volume of fluid can also benefit from one or more accumulators. These are often placed close to the demand to help over comerestrictions and drag from long pipe work runs. The outflow of energy from a discharging accumulator is much greater, for a short time, than even large pumps could generate.

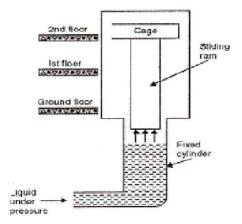
An accumulatorcan maintain the pressure in a systemfor periods when there are slight leaks without the pump being cycled on and off constantly. When temperature changes cause pressure excursions the accumulator helps absorb them. Its size helps absorb fluid that might otherwise be locked in a small fixed system with no room for expansion due to valve arrangement.



DirectActingHydraulicLift:

CONSTRUCTIONDETAILS:

- I fixedcylinder:Itisfixedwiththewallofthefloor,wheretheslidingram reciprocate when weapply the pressure.
- © Cage:It is fitted on the top of the sliding ram where the load is placed (i.e. lifted load).
- Slidingram:Itisfittedinthefixedcylinderwhichisreciprocate(upwardordownward direction)when we applied the pressure (i.e. reaches the floor wise.)



When fluid under pressure is forced into the cylinder, the ram gets a push upward. The platformcarries loads or passengers and moves between the guides. At required height, it can be made to stay in level with each floor so that the good or passengers can be transferred.

Indirectactinghydrauliclift, strokeoftheramisequal to the lift of the cage.

SUSPENDEDHYDRAULICLIFT

CONSTRUCTIONDETAILS:

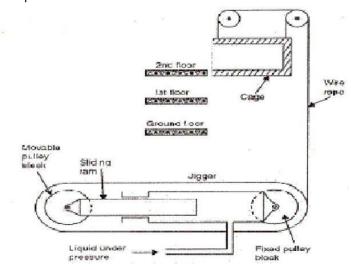
Cage:Itisfittedonthetop of thesliding ramwherethe loadisplaced (i.e.lifted load). Wire rope: It connects the cage to pulley.

Sliding ram: It is fitted in the fixed cylinder which is reciprocate (upward or downward direction)when we applied the pressure (i.e. reaches the floor wise)

Pulleys: pulleys are connected to the sliding ram and fixed cylinder; where one pulley is fixed and other pulley is movable.

Hydraulicjigger:Itconsistsofamovingramwhichslidesinsideafixedhydrauliccylinder.

Fixed cylinder-: It is fixed with the wall of the floor, where the sliding ram reciprocate whenweapply the pressure.



WORKINGOFSUSPENDEDHYDRAULICLIFT

When fluid under pressureis forced into the cylinder, the ramgets reciprocate to the movable pulleys. With the helpofarrangement of hydraulic jigger; pulley can rotates; with the help of

wireropethecage is maintainthepressureforcewith their floor. At required height, it can be made to stay in level with each floor so that the good or passengers can be transferred.

Workingperiodoftheliftis ratiooftheheightoflifttothevelocityoflift.

Idle period of lift is the difference of the total time for one operation and the working period of the lift.

Hydraulicram

Thisarticle isaboutthe waterpump. Forthe vehicle extractiontool, see Hydraulic rescue tools. Forthe piston-based actuator, see hydraulic cylinder.

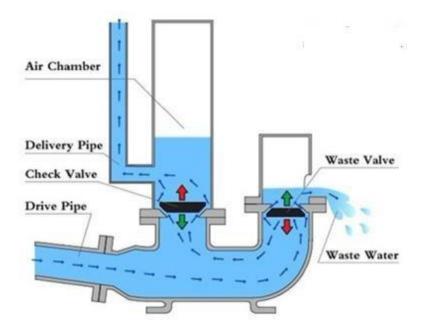
A hydraulicram, is a cyclicwaterpump poweredby hydropower. Ittakes inwateratone "hydraulichead" (pressure) and flow rate, and outputs water at a higher hydraulichead

and lower flow rate. The device uses the water hammer effect to develop pressure that allows a portion of the input waterthat powers the pump to be lifted to apoint higher than where the water originally started. The hydraulic ram is sometimes used in remote areas, where there is both a source of low-head hydropower and a need for pumping water to a destination higherin elevation than the source. In this situation, the ram is often useful, since it requires no outside source of power other than the kinetic energy of flowing water.

The working principle of hydraulic ram is to use surge pressure which is produced after flow blocked and ten times higher than normal to lift water.

Before working, waste valve stays open under the action of magnet spring while delivery valve keep closed under the action of magnetspring andits gravity. Itcanwork automaticallywhenwe controlthe waste valve to repeat the operation proceduresofopen and close. Afterthat, waterwithdifferent levelswillflowoutthroughwaterdrivepipeandopened waste valve, and running water will drive the waste valve to close when the pressure insidethe waste valve surpass that in magnet spring, and that is the water hammer. At the moment, water pressure rapidly increases and enforces the delivery valve to open, and some water flows into air chamber. Pressure inside the waste valve drops promptly and the waste valve reopens underthe action of magnet spring and negative pressure. While delivery valve closes again by the action of self gravity and the pressure magnetspring and air chamber. By the

actionofwaterflow,movementsforegoingrepeatautomatically. Andwaterwill flowout through the delivery pipe when the pressure in air chamber exceeds that in lifting pipes.



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